



# FFlex Mesher Suspension Tutorial (Durability)

**Durability Contour**

Contour Option: Safety Factor

Band Option: Legend Type: Display, Location: Bottom, Show Text Legend: , Band Level(10-50): 10

Min/Max Option: Display: , Calculation: Min: 108.12, Max: 562.38, Show Min/Max: , Enable Log Scale:

Style Option: Color Option: Edit, Colors: Spectrum, Style: Smooth, Text Color: [Yellow], Mesh Lines:

**User-Defined**

No	Cycle to Failure	Stress Amplitude
1	1.	820.
2	10.	650.
3	100.	470.
4	1000.	360.
5	10000.	270.
6	100000.	200.
7	1000000.	150.
8	10000000.	120.
9	100000000.	90.

**History**

Original History: History\_1

Stress vs Time plot showing a sinusoidal wave oscillating between approximately 1.0 and -1.0.

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**Edition Note**

This document describes the release information of **RecurDyn V9R4**.

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## Chapter

## 1

## Introduction

Fatigue and durability analyses are designed to determine how long a flexible body or specific area of a flexible body modeled in RecurDyn can stably endure various dynamic loads. Such analyses can also determine how stable a body is. The focus on time distinguishes these forms of analysis from other analysis methods, such as those used to determine maximum stress and maximum deformation rates.

This tutorial teaches you how to use **RecuDyn/Durability** to determine the fatigue life and fatigue damage in a dynamic system composed of flexible bodies. This tutorial also briefly describes functions and theoretical backgrounds used in RecurDyn/Durability. Thus, even users who do not have a thorough grounding in the theoretical background of durability analysis can learn to use RecurDyn/Durability simply by completing this tutorial.

This tutorial uses the MBD model, which features a simplified system intended for experimental purposes. By following the steps in this tutorial, you will learn how to replace specific sections of the flexible bodies in the model using RecurDyn/Mesher and verify the properties and durability results of these flexible bodies. Later in this tutorial, you will learn how to produce durability results for a specific section of the model.

## Task Objectives

This tutorial covers the following:

- Creating a flexible body using RecurDyn Mesher
- Requirements for durability analysis
- Methods of conducting durability analysis
- Obtaining durability analysis results
- Analyzing durability analysis results

## Requirements

This tutorial is intended for intermediate users who have read and understood the basic tutorial as well as the FFlex and RFlex tutorials available from RecurDyn. If you have not completed these tutorials, then you are advised to complete them before proceeding with this tutorial. In addition, this tutorial requires a basic understanding of dynamics and the finite element method.

## Tasks

This tutorial consists of the tasks outlined in the following table. This table also shows the time required to complete each task.

<b>Procedures</b>	<b>Time (minutes)</b>
Calling an Rdyn model	10
Creating an FFlex body using Mesher	15
Replacing an RFlex body	20
Creating a patch set to verify the fatigue results	5
Setting fatigue preferences	5
Performing a fatigue evaluation	5
Verifying the fatigue results	5
<b>Total</b>	<b>65</b>



### **Estimated Time to Complete**

This tutorial takes approximately 65 minutes to complete.

**Chapter**

**2**

## Calling the Initial Model

### Task Objective

This chapter teaches you how to open an initial model, simulate it, and observe the behavior of the suspension model.



### **Estimated Time to Complete**

10 minutes

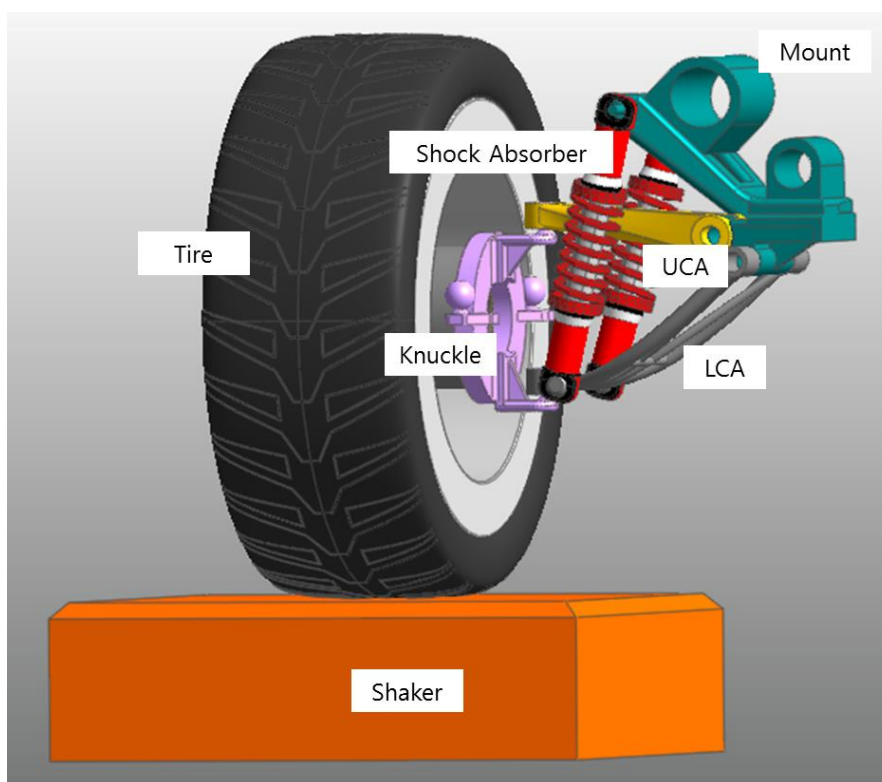
## Calling a Rdyn Model

To open RecurDyn and call the initial model:



1. On the desktop, double-click the **RecurDyn** icon.
2. When the **Start RecurDyn** dialog window appears, close it.
3. From the **File** menu, click **Open**.
4. From the **Durability** tutorial path, select the **RD\_Durability\_Start.rdyn** file. (**The file location:** <Install Dir>\Help\Tutorial\PostAnalysis\Durability\FFlexMesherSuspension, ask your instructor for the location of the directory if you cannot find it).
5. Click **Open**.

The model shown below opens.



This model is designed to test the durability of the parts in a suspension system. A suspension system absorbs shocks from street surfaces in order to prevent them from being transferred to the vehicle or passengers. In this model, the tire rests on the shaker so that any excitations pass directly from the shaker to the tire and the suspension system.

The suspension system in this model consists of one tire and three subsystems. The three subsystems include two shock absorbers with built-in coil springs and a suspension assembly. The suspension assembly consists of a mount, a knuckle, an LCA (lower connecting arm), and a UCA (upper connecting arm). In this tutorial, you will test the durability of the LCA and the UCA.

**To save the initial model:**

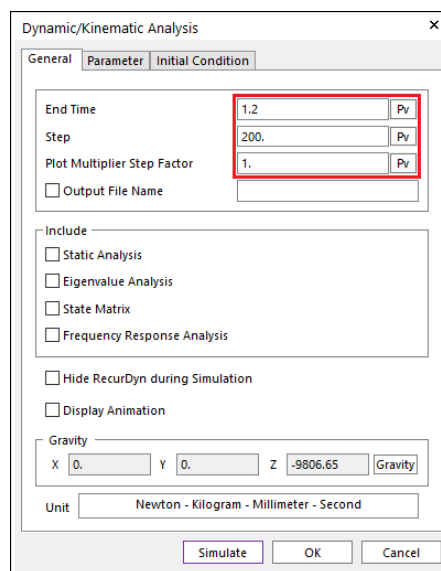
- From the **File** menu, click **Save As**.  
(Save this model in the different path because it is impossible to simulate directly in the tutorial path.)

**Running an Initial Simulation with the Suspension Model**

In this task, you will run an initial simulation on the model to understand how it operates.

**To run an initial simulation:**

- From the **Simulation Type** group in the **Analysis** tab, click **Dyn/Kin**.  
The **Dynamic/Kinematic Analysis** dialog window appears.
- Define the **End Time** and **Step** values as follows:
  - End Time:** 1.2
  - Step:** 200
  - Plot Multiplier Step Factor:** 1
- Click **Simulate**.

**Viewing the Results****To view the results:**

From the **Animation Control** group in the **Analysis** tab, click **Play/Pause**.

The excitation is transferred to the tire through the shaker. The animation shows any possible vibrations or seismic excitation caused by driving the vehicle. At this point, you can observe the behavior of certain devices, including the shock absorbers.



## Chapter

## 3

## Creating an FFlex Body

This chapter teaches you how to conduct a durability analysis on a flexible body. This procedure maintains some elements of the previous model, such as its joint and force, but replaces certain rigid bodies with flexible bodies to increase the efficiency of the model.

### Task Objective

This task teaches you how to replace an existing rigid body with a flexible body using the Mesher provided in RecurDyn FFlex (Full Flex).

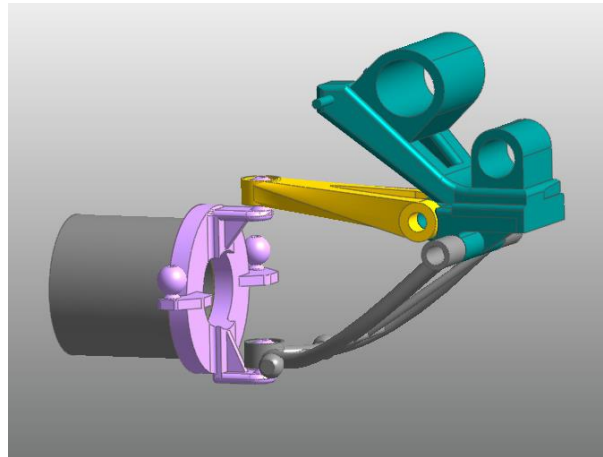
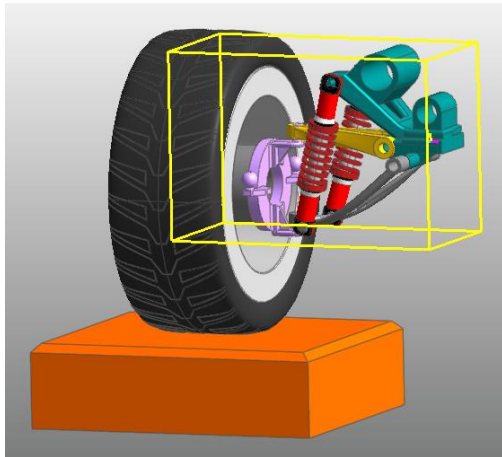


### Estimated Time to Complete

15 minutes

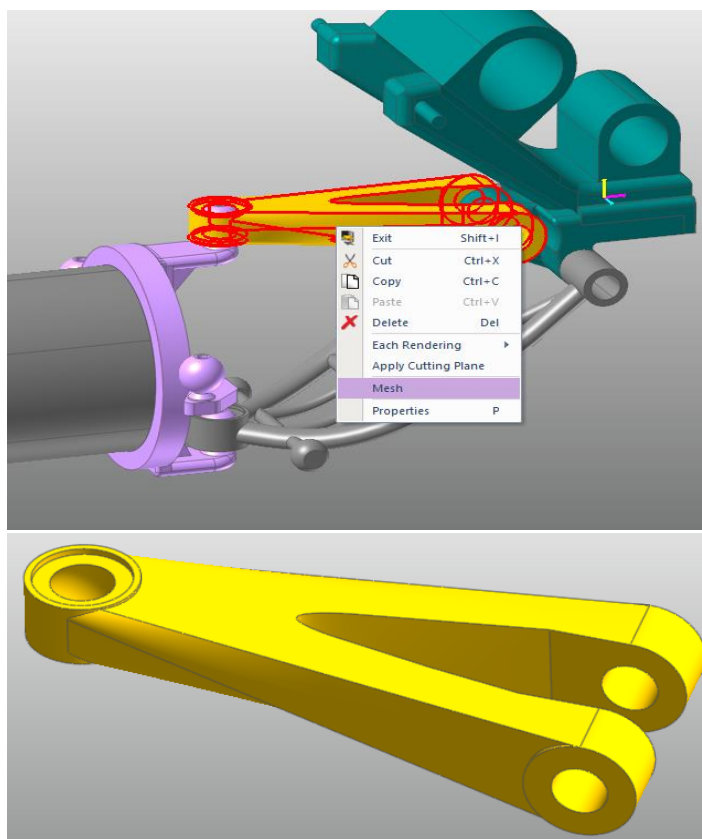
## Creating a UCA Body Mesh

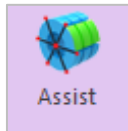
1. In **Assembly** mode, double-click the **Suspension\_Assy** created as a Subsystem to enter the **Subsystem** mode.



2. In **Subsystem** mode, select the **UCA** body and Right-click the selected body, and select **Mesh** from the context menu, as shown in the figure to the right.

The UCA body appears as shown in the figure below.





- In **Mesh** mode, click **Assist** from the **Mesher** group in the **Mesher** tab.

The **Assist Modeling** dialog window appears.

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**Tip:** The **Geometries** in the **Preserve Constraint** are applied to the **Assist Modeling** dialog automatically after the user selects the **Target Body**. However, following behind steps are recommended to the user to produce accurate results, especially in the tutorial.

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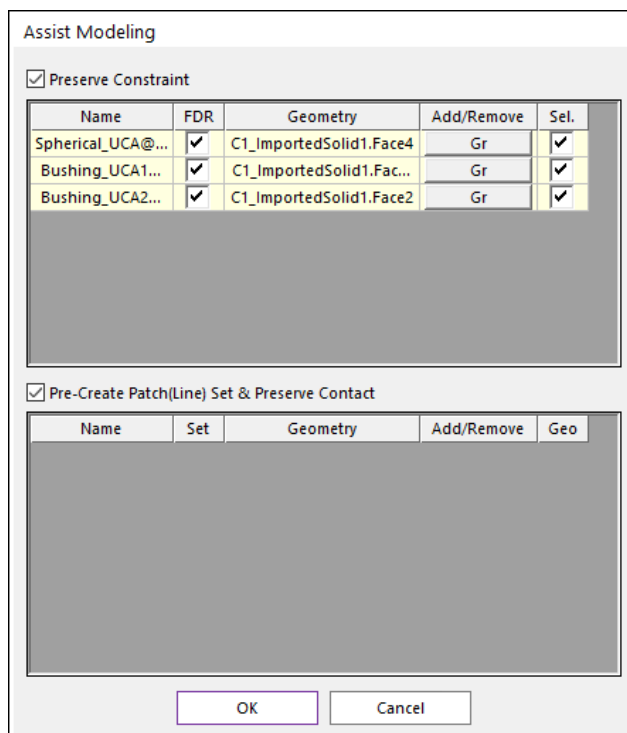


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**Tip:** The FDR checkboxes specify whether or not to create an FDR (Force Distributed Rigid) element after the Mesh operation. The Sel. checkboxes specify whether or not to maintain the joint or force previously created. In this tutorial, these functions are all necessary. Select all of the checkboxes.

---

- In the **Add/Remove** column of the dialog window, click **Gr** to assign the geometries for the areas where the **Slave Nodes** will be created. Referring to the following page, select the geometries where the slave nodes will be created for the three FDRs. (**Tip:** When you click the **Gr** button, a marker appears on the screen indicating where the master node will be generated. This helps you locate the areas to assign slave nodes.)

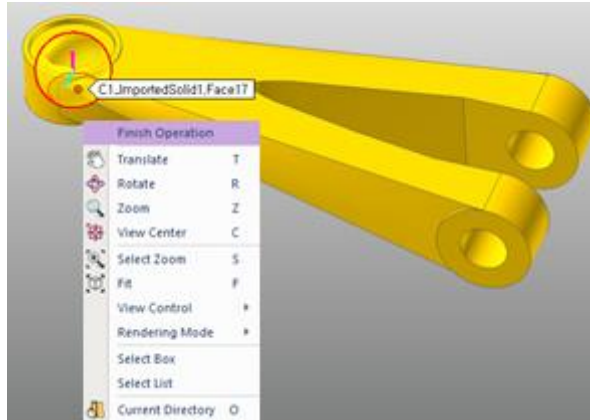



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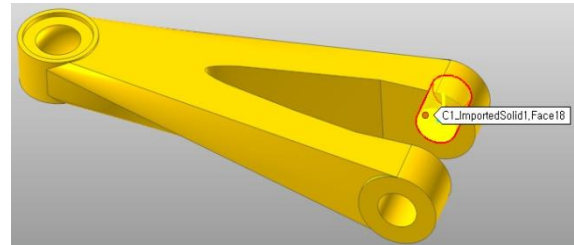
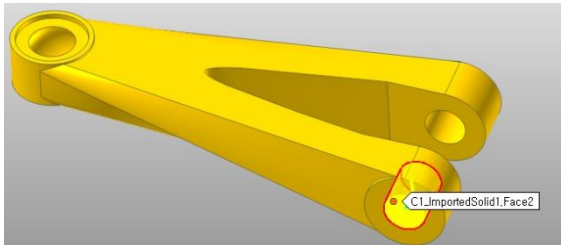
**Tip:** Due to the nature of the FDR, the master node is automatically generated at the point where the joint and the force are created. You must define the slave nodes directly.

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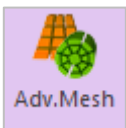
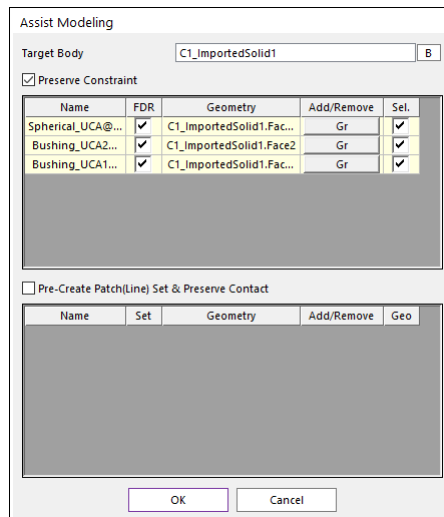
- After all the geometries are assigned, right-click the model, and click **Finish Operation**.



6. For the other two areas, select the **Surfaces** where the **Slave Nodes** will be generated, as shown in figures below.

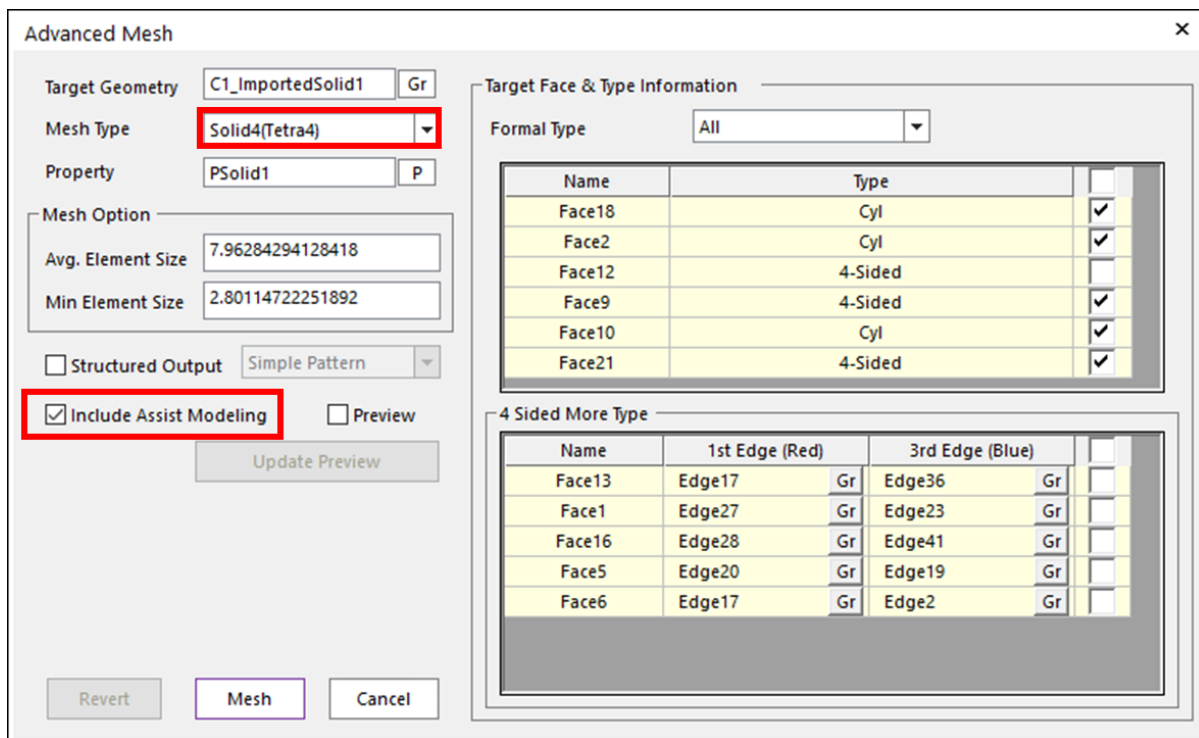


7. Once you have selected all the geometry items, click **OK**.



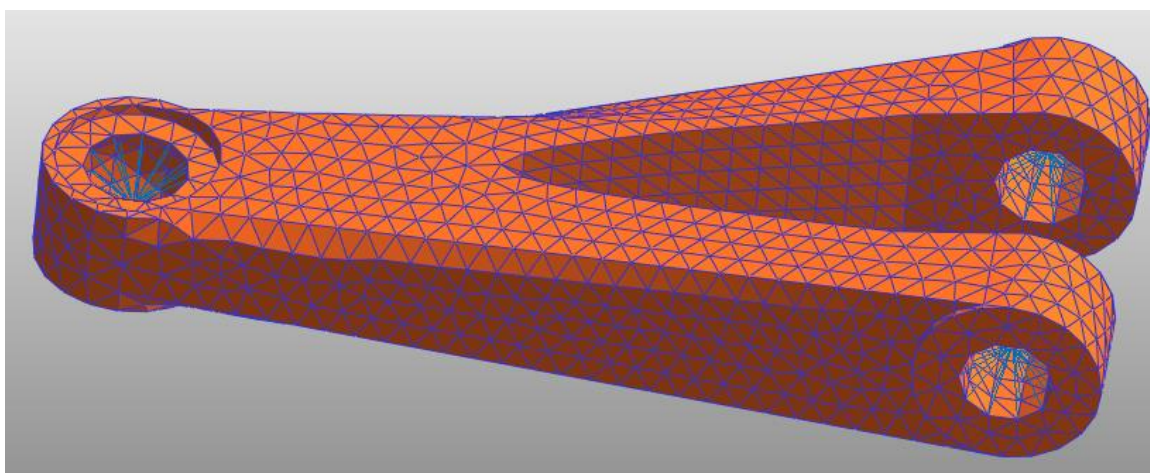
8. From the **Mesher** group in the **Mesh** tab, click **Adv.Mesh**.

The Advanced Mesh dialog window appears.



9. In the **Advanced Mesh** dialog window, perform the following:
  - In the **Mesh Type** dropdown menu, select **Solid4 (Tetra4)**.
  - Select the **Include Assist Modeling** checkbox. (**Tip:** You must select this to ensure that the FDRs are automatically generated according to the conditions defined in Assist Modeling.)
  - Click **Mesh**.
  - Once Meshing is complete, click **Close**.

The mesh model of the UCA body appears, as shown below.

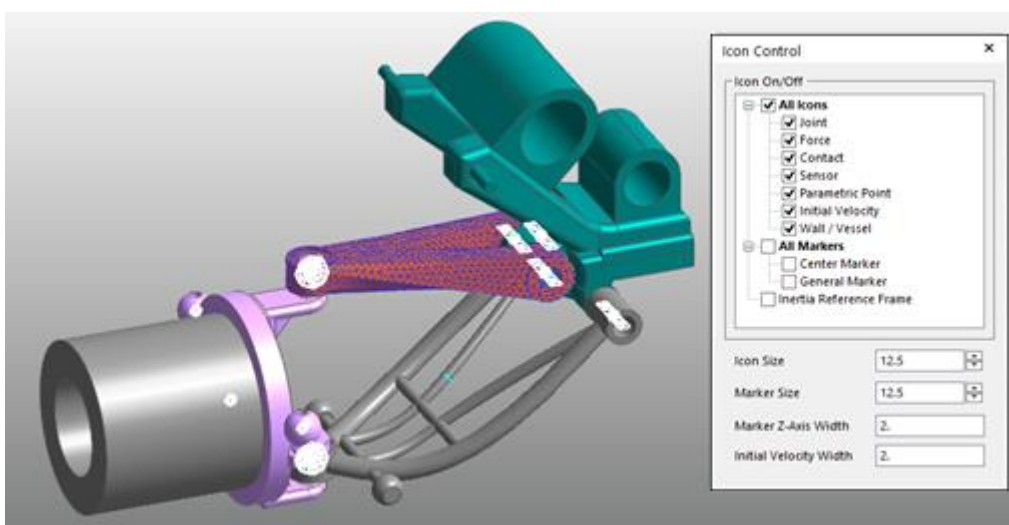


**Tip:** RecurDyn offers two auto-meshers, Mesh and Advanced Mesh. The Mesh auto-mesher applies tessellation to the geometry in the GUI before generating the mesh. The Advanced Mesh auto-mesher skips the tessellation stage altogether and generates the mesh directly. By doing so, Advanced Mesh minimizes the distortion of the CAD geometry. However, Advanced Mesh also increases the Mesh failure rate.

10. Perform either of the following steps to return to the parent mode:

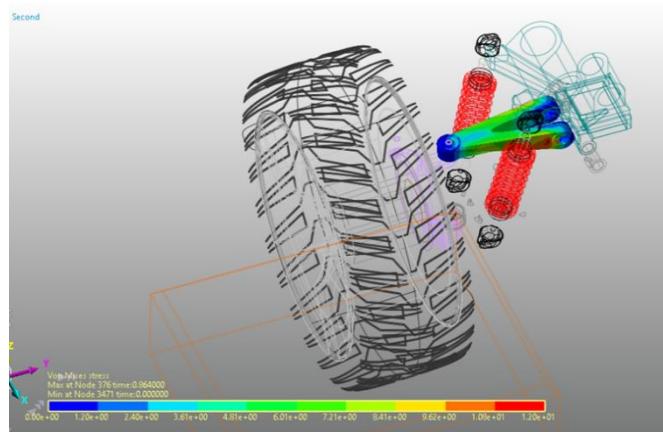
- Right-click the Working window, and from the menu that appears, click **Exit**.
- From the **Exit** group in the **Mesher** tab, click **Exit**.

Note that the existing UCA Rigid Body (UCA) disappears and it is replaced by the UCA Flexible body (UCA\_FE), as shown in the figure below. Also, the joint and the force previously created are maintained as they were in the GUI. At this moment, to see the icons, click **Icon Control** on the toolbar and select **All Icons**.



11. To check if the modeling is correctly configured for the **UCA Flexible body**, return to the **Assembly** mode and run **Dynamic Analysis** using the previous conditions. The analysis will be completed quickly.

12. From the **FFlex** group in the **Flexible** tab, click **Contour** to view the results.

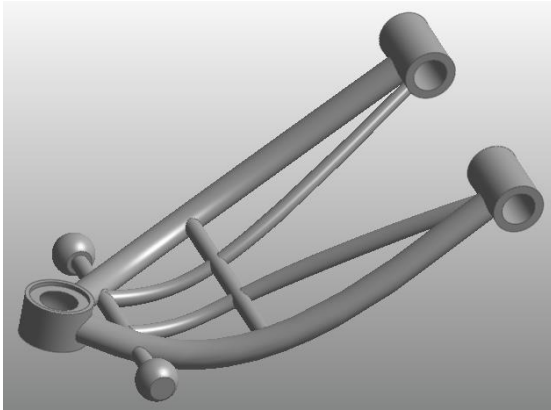


## Creating an LCA Body Mesh

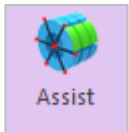
1. In **Assembly** mode, double-click the **Suspension\_Assy** created as a subsystem to enter the **Subsystem** mode.

In **Subsystem** mode, select the body and the **LCA** body. Right-click the selected body, and select **Mesh** from the context menu, as shown in the figure to the right.

2. The LCA body appears as shown in the figure below.

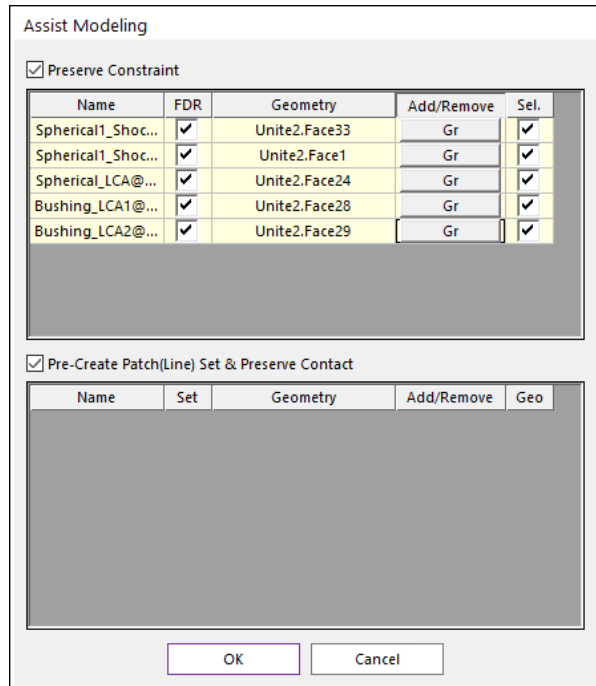


3. In **Mesh** mode, click **Assist** from the **Mesher** group in the **Mesher** tab.



The **Assist Modeling** dialog window appears.

- In the Add/Remove column of the dialog window, click **Gr** to assign the geometries for the areas where the Slave Nodes will be created. Referring to the following page, select the geometries where the slave nodes will be created for three FDRs. (**Tip:** When you click the **Gr** button, a marker appears on the screen indicating where the master node will be generated. This helps you locate the areas to assign slave nodes.)

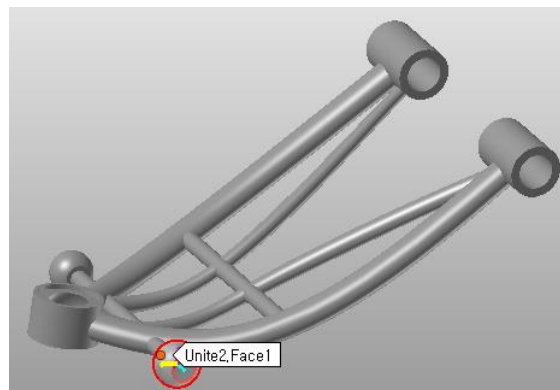



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**Tip:** Due to the nature of the FDR, the master node is automatically generated at the point where the joint and the force are created. You must define the slave nodes directly.

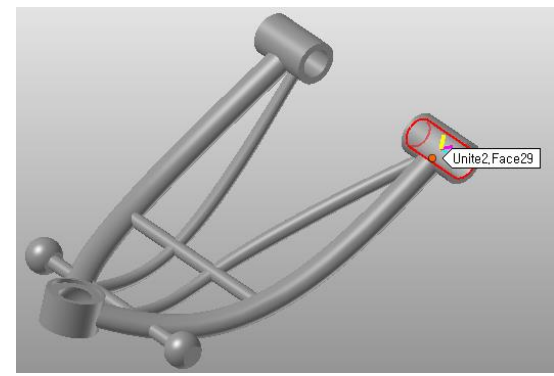
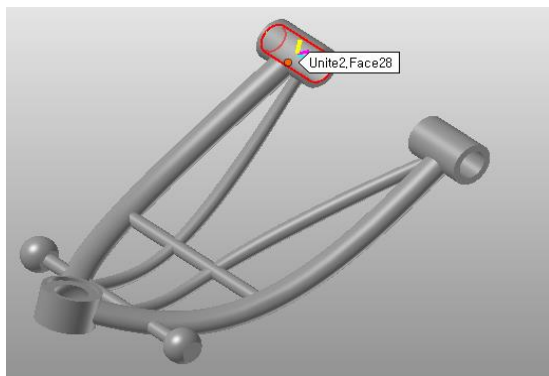
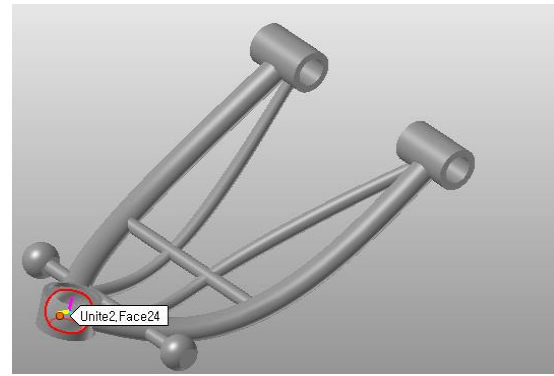
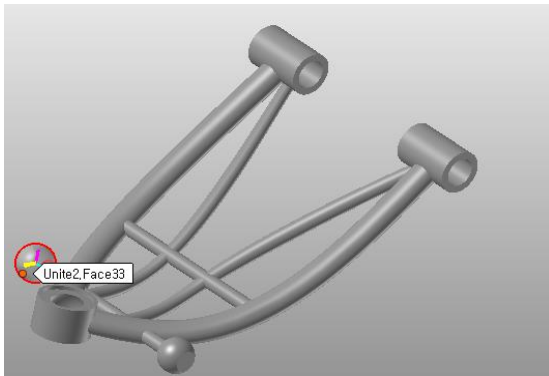
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- After all the geometries are assigned, right-click the model, and click **Finish Operation**.

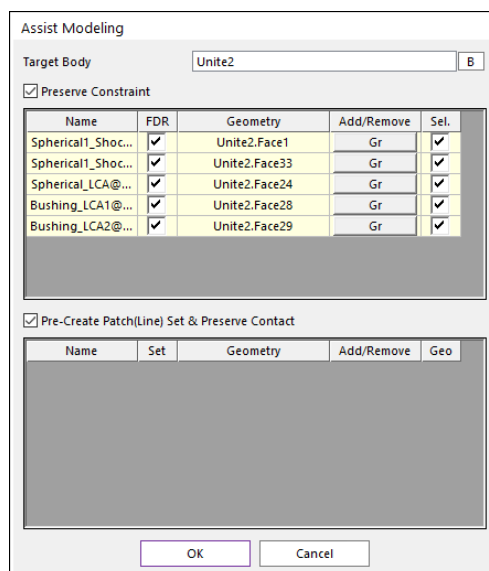




6. For the other four areas, select the **Surfaces** where the **Slave Nodes** will be created, as shown in the figures below:



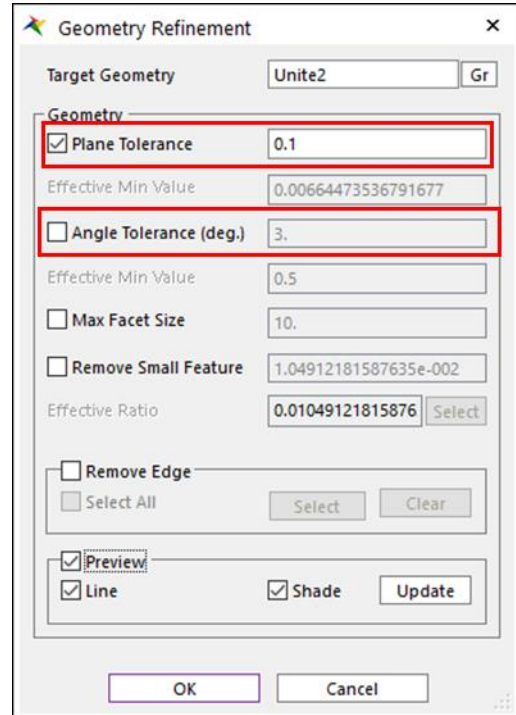
7. Once you have selected all the geometry items, click the **OK** button.





- From the **Mesher** group in the **Mesher** tab, click **Geo. Refine**. (**Tip:** The geometry refinement function is only effective when using the mesh method. That is, as with the UCA body, if the advanced mesh method is used, then it has no effect at all.)

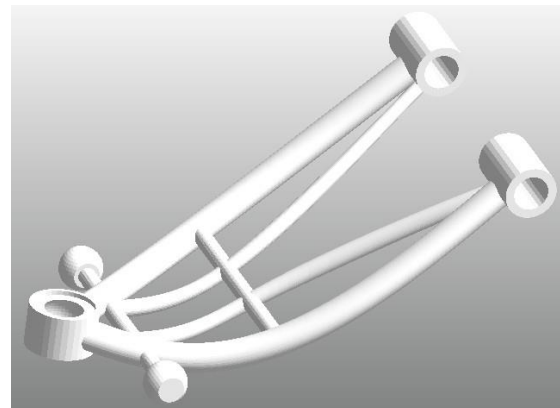
The Geometry Refinement dialog window appears.



- In the Geometry Refinement dialog window, change the **Plane Tolerance** to **0.1**.
- In the Geometry Refinement dialog window, check out the **Angle Tolerance**
- Select the **Preview** checkbox.

The tessellated shape appears, as shown in the figure to the right.

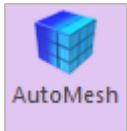
- Click the **OK** button to close the dialog window.




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**Tip:** RecurDyn features two Auto-Meshers, Mesh and Advanced Mesh. The Mesh auto-mesher applies tessellation on the geometry in the GUI before generating the mesh. The Advanced Mesh auto-mesher skips the tessellation stage altogether and generates the mesh directly. The LCA body has a lot of curved surfaces, so you should use the Mesh auto-mesher (which applies tessellation) to reduce the number of nodes. The geometry refinement function used for the LCA body applies the tessellation and has nothing to do with the Advanced Mesh method.

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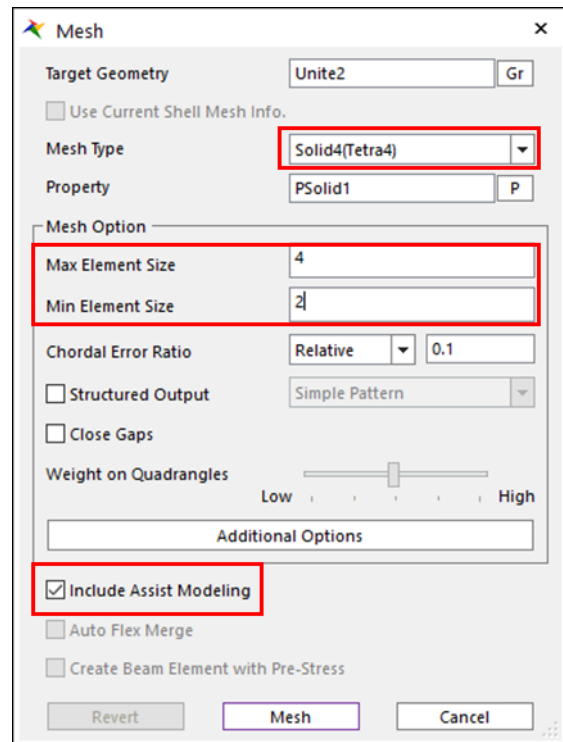


13. Click **AutoMesh**.

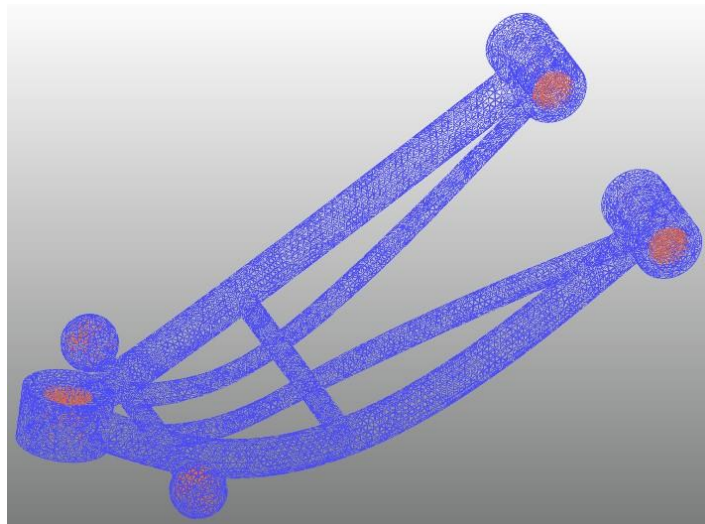
The **Mesh** dialog window appears.

14. In the **Mesh dialog** window, perform the following:

- Select **Solid4 (Tetra4)** as the **Mesh Type**, as shown in the figure to the right.
- In the Mesh Option group, set the **Max Element Size** to **4** and the **Min Element Size** to **2**.
- Select the Include Assist Modeling checkbox.
- Click the **Mesh** button.



The mesh model and FDR of the LCA body are created, as shown in the figure below.

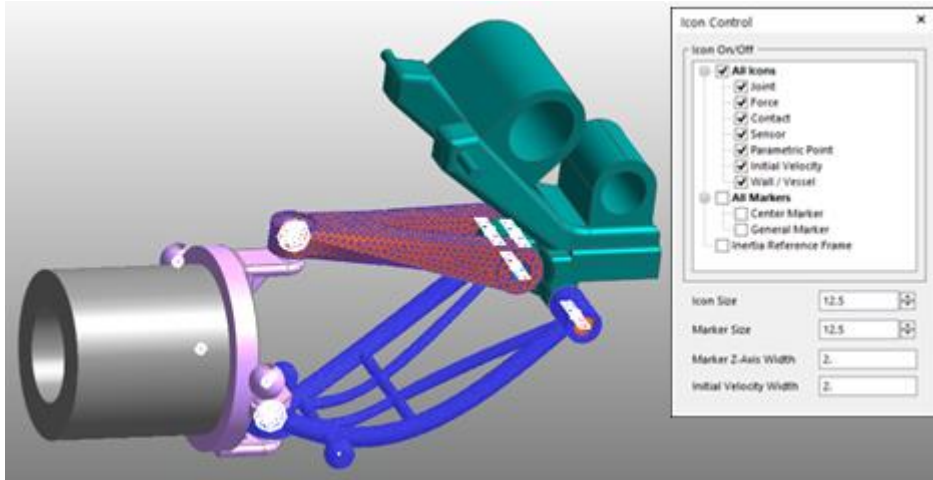


15. Perform either of the following steps to return to the parent mode:

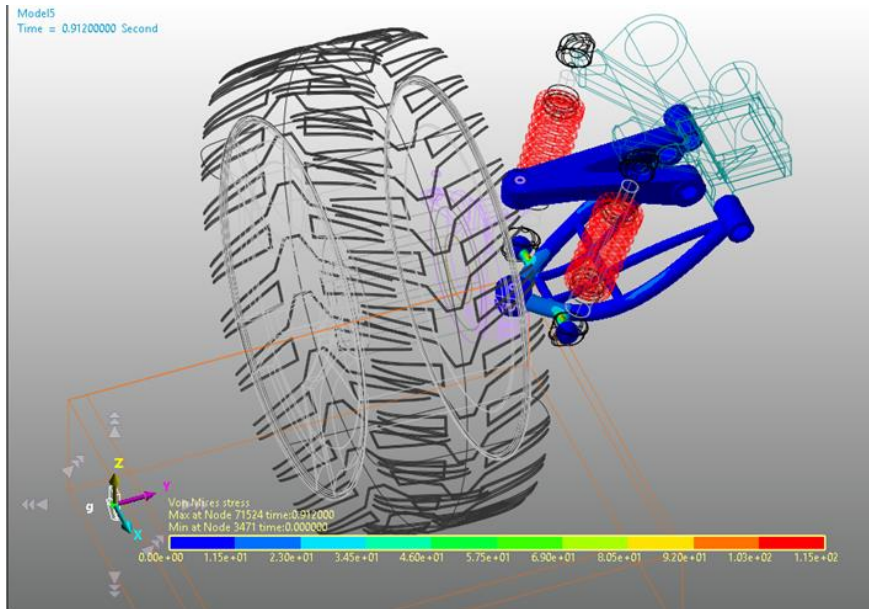
- Right-click the Working window, and from the menu that appears, click **Exit**.
- From the **Mesher** group in the **Mesher** tab, click **Exit**.



Note that the existing **LCA Rigid Body (LCA)** disappears and it is replaced by the **LCA Flexible body (LCA\_FE)**, as shown in the figure below. Also note that the joint and the force previously created are maintained from the GUI. At this point, to see the icons, click **Icon Control** on the toolbar and select the **All Icons** checkbox.



- To check if the modeling is correctly configured for the created **LCA Flexible body**, **Exit** to the **Assembly** mode and run the **Dynamic Analysis** under the previous conditions. The analysis will take a little longer than in the previous step, but it will not take too long to produce the results. From the **FFlex** group in the **Flexible** tab, click **Contour** to produce the results shown below.



**Chapter**

**4**

## **Conducting Durability Analysis**

### **Task Objective**

This chapter teaches you how to analyze the durability of an FFlex body.



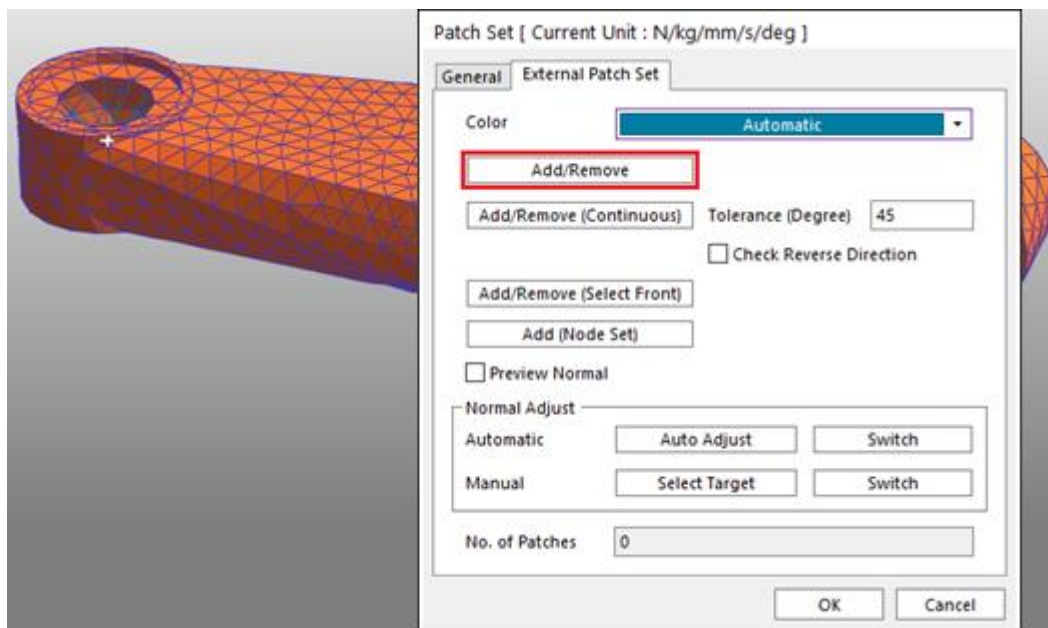
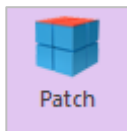
### **Estimated Time to Complete**

40 minutes

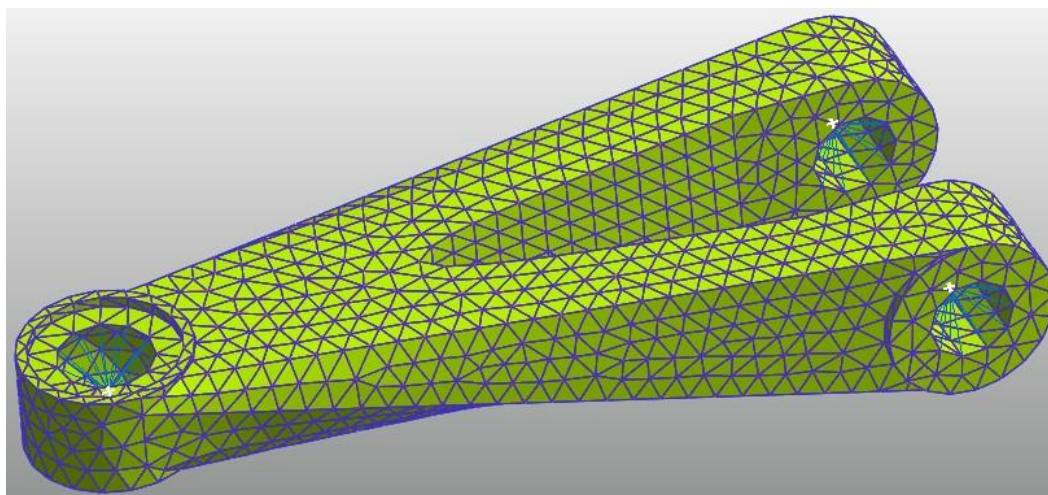
## Conducting a Durability Analysis on the UCA\_FE Body

To create a patch set:

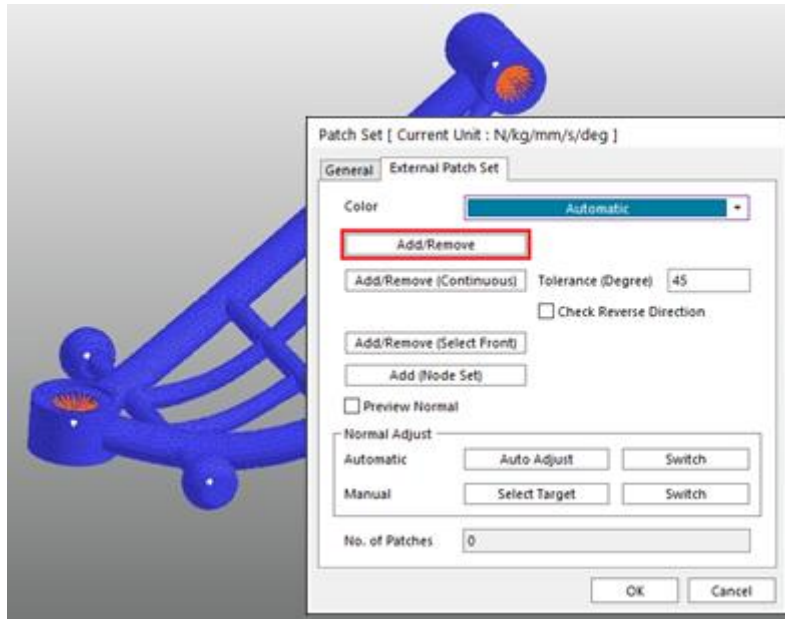
1. In **Assembly** mode, double-click the **Suspension\_Assy** to enter **Subsystem** mode.
2. Double-click the **UCA\_FE** body to proceed to the **Body Edit** mode.
3. From the **Set** group in the **FFlex Edit** tab, click **Patch Set**.
4. Click the **Add/Remove** button, as shown below. Click and drag the mouse button to select the entire body.



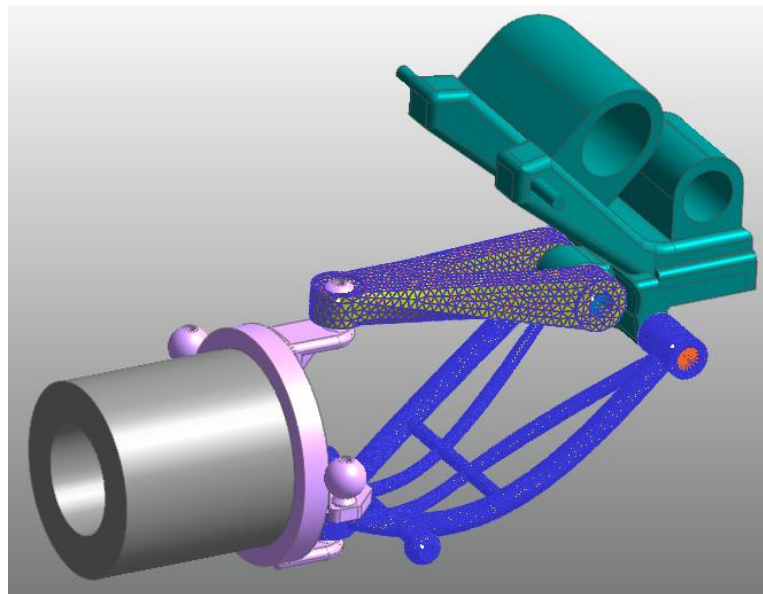
5. Right click the body and click **Finish Operation** in the context menu.
6. In the Patch Set dialog window, click **OK**.
7. Confirm that the patch set is created and click **Exit** from the **Exit** group in the **FFlex Edit** tab to return to the parent mode.



- To set the patch set for the **LCA\_FE** body, repeat steps 2 to 7, as shown in the figure below.



When you have completed the above procedures, the patch sets have been created for each of the two Flexible Bodies.



- Right-click the Working window, and from the menu that appears, click **Exit** to return to the **Assembly** mode.

**To retrieve the animation file:**

From the **Animation Control** group in the **Analysis** tab, click **Reload the last animation file** icon.

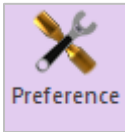
Note that all of animation-related buttons are activated.

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**Tip:** After you create the patch sets for the UCA\_Fe and LCA\_FE bodies, it may look as if the previous analysis results are unavailable. However, those procedures have no effect on any of the dynamic analysis results. Therefore, there is no need to perform dynamic analysis again. You can simply retrieve the animation file, or the RAD file, from the previously analyzed results.

---



**To set the analysis preferences:**

1. From the **Durability** group in the **Post Analysis** tab, click **Preference**.
2. In the Preferences dialog window, on the **Material** tab, specify the path of the **Material Library file** for the Fatigue Analysis.  
(C:\Users\<<Your Windows Login ID>\Documents\RecurDyn\<RecurDyn Version>\Durability or the equivalent path).
3. On the **Fatigue Influencing Factors** tab, in the Fatigue Factors group, set the **Notch Factor Amp (Kf, Kt)** to **2**, as shown below.

The screenshot shows the 'Preferences' dialog box with the 'Fatigue Influencing Factors' tab selected. Under the 'Fatigue Factors' section, the 'Notch Factor Amp (Kf, Kt)' field is highlighted with a red box and contains the value '2'. Other fields include 'Surface Factor (ms)' with a dropdown set to 'Polished' and a value of '1.0', 'Size Factor (md)' set to '1', 'Load Factor (mt)' set to '1', and 'Other Factor (mo)' set to '1'. Under the 'Overall Scale Factor' section, the 'Scale Factor (fs)' field is set to '1'. At the bottom of the dialog are 'OK', 'Cancel', and 'Apply' buttons.

The **Notch Factor** increases the analytically derived stress value to account for stress concentrations due to the cracks, holes, and notches (V grooves) caused by the design and processing of the structure. Therefore, the larger the notch factor value is, the more severe the durability analysis will be.

**Tip:** For reference, the values included in the Fatigue Influencing Factors represent the physical conditions of actual test samples in an actual durability analysis experiment, so they all have default values of 1. Modifying each factor varies the S-N curve accordingly: making it possible to simulate more severe conditions in the durability analysis. If the values other than the notch factor are less than 1 and the notch factor value is greater than 1, then the conditions simulated in the durability analysis will be more severe.

4. In the **Preferences** dialog window, do not change the Convergence Control and **Rainflow Counting** values, and click **OK**.

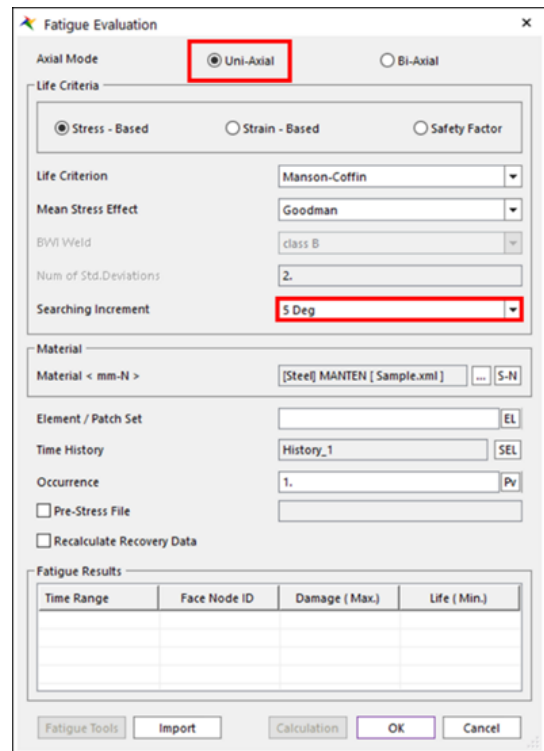
**To conduct the fatigue evaluation:**



1. From the **Durability** group in the **Post Analysis** tab, click **Fatigue Evaluation**.

The Fatigue Evaluation dialog window appears.

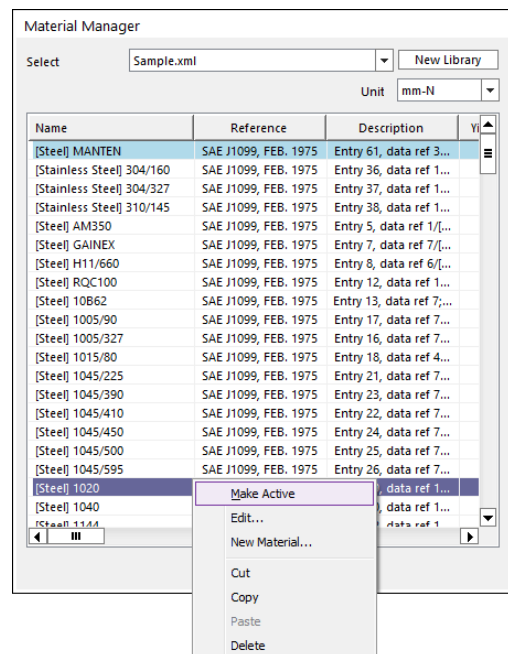
2. In the **Fatigue Evaluation** dialog window, perform the following:
  - Set the **Axial Mode** to **Uni-Axial**.
  - Set the **Searching Increment** to **5 Deg**.



3. In the Material group, click "...".

The Material Manager dialog window appears.

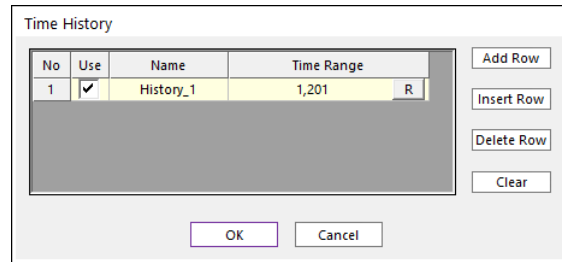
4. In the **Material Manager** dialog window, perform the following:
  - Select and right-click **[Steel] 1020**, and click **Make Active** in the context menu, as shown in the figure to the right.
  - Click **OK**.



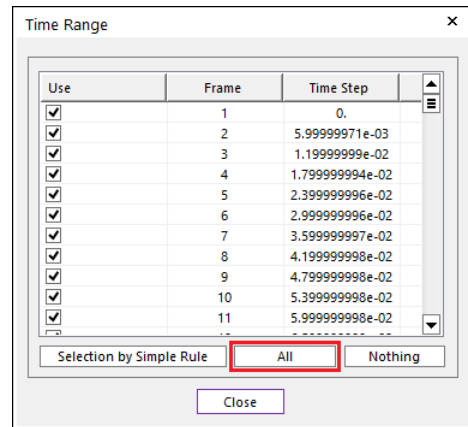
5. In the **Element/Patch Set** group, click **EL**.
6. Select the set **Patch Set** for the **UCA\_FE** body.

Since the **UCA\_FE** body is included in the **Suspension\_Assy** subsystem, you must click the **UCA\_FE** body button while pressing the **Shift Key**. As a result, **UCA\_FE.SetPatch1@Suspension\_Assy** will appear on the Element/Patch Set item.

7. Click the **SEL** button in the **Time History** text box.
8. A Time History set is defined already, in the **Time History** dialog window. To change the range of time, click **R**.
9. In the **Time Range** dialog window, click **All**.

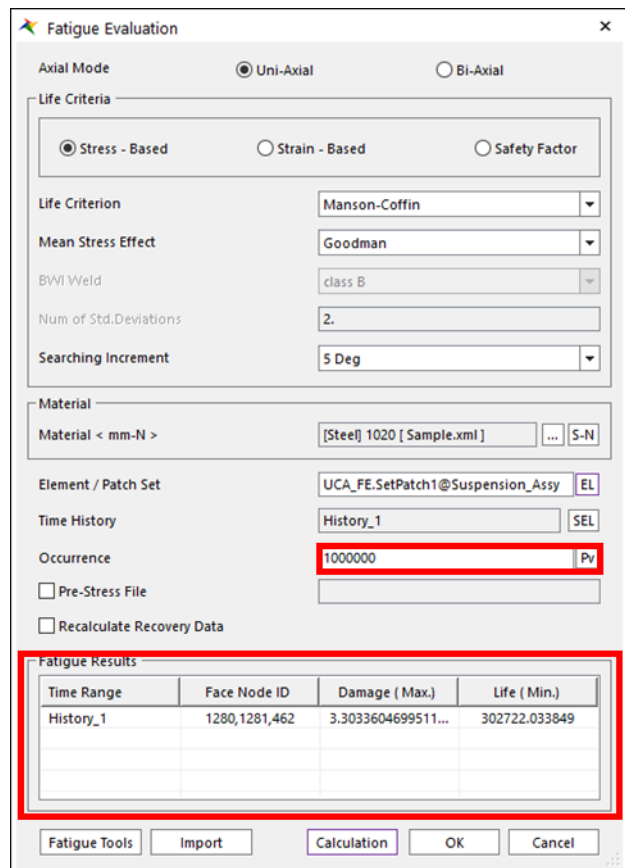


10. Click **Close** in the **Time Range** dialog window.
11. Click **OK**



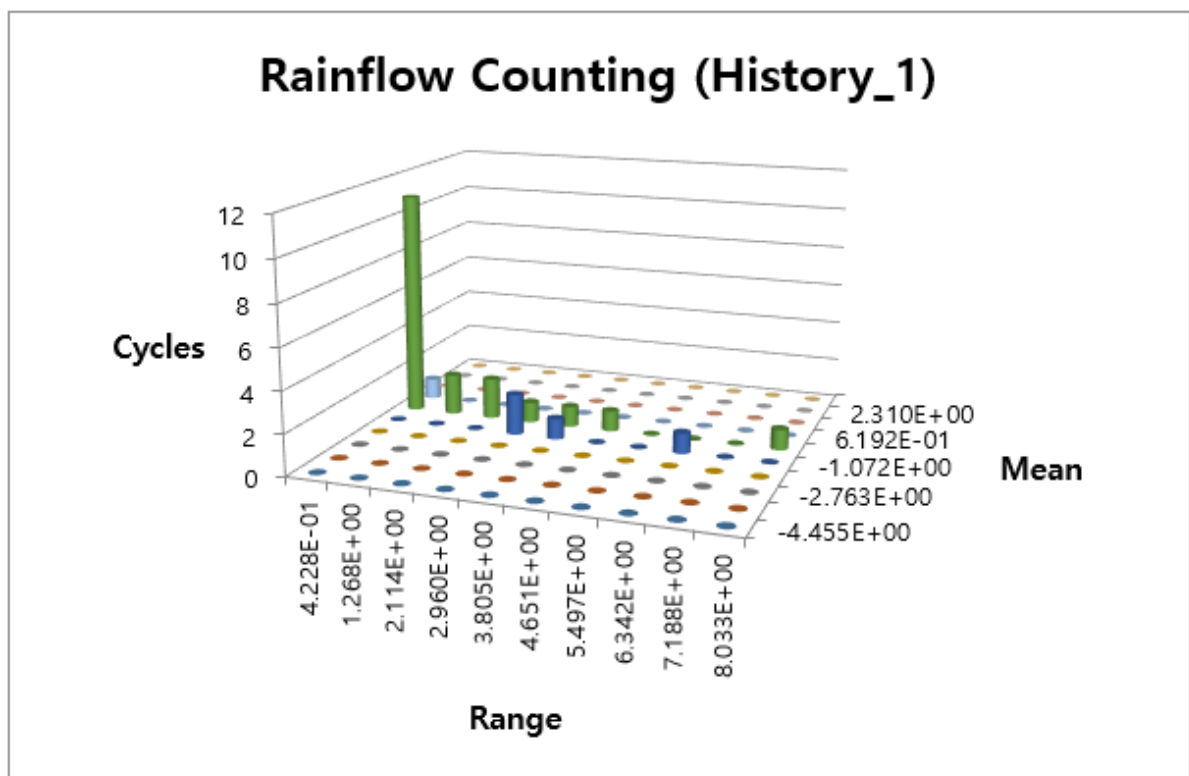
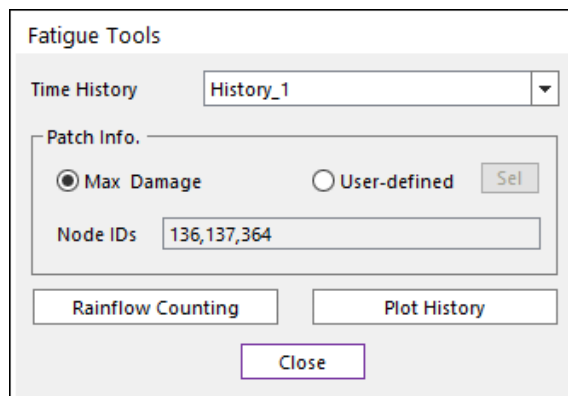
12. In the **Occurrence** text box, type **1000000**.
13. Click **Calculation**.

A Progress Bar appears to display the fatigue analysis progress. Upon completion of the analysis, the results will appear in the results group, as shown in the figure to the right.

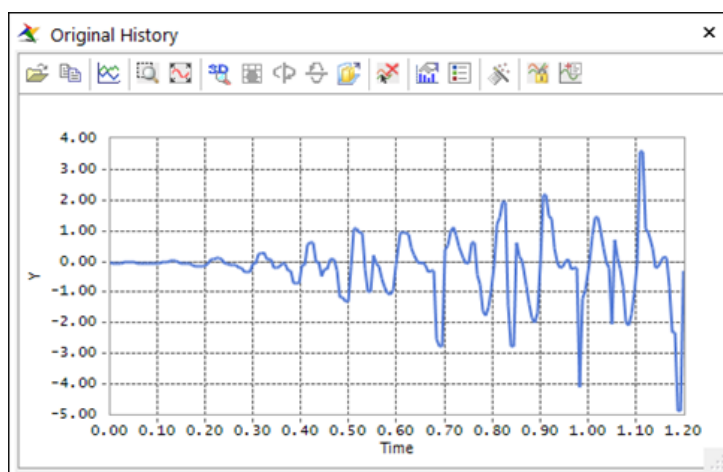


14. In the Fatigue Evaluation dialog window, click **Fatigue Tools**.

- Click the **Rainflow Counting** button in the Fatigue Tools dialog.
- As shown below, the Rainflow Counting results in the Excel are based on the Stress Time History applied to the patch zone where the damage is largest. The results are displayed in the numbers of cycles according to the stress amplitude and the mean stress.



- Click the **Plot History** in this dialog.



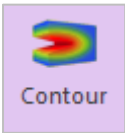
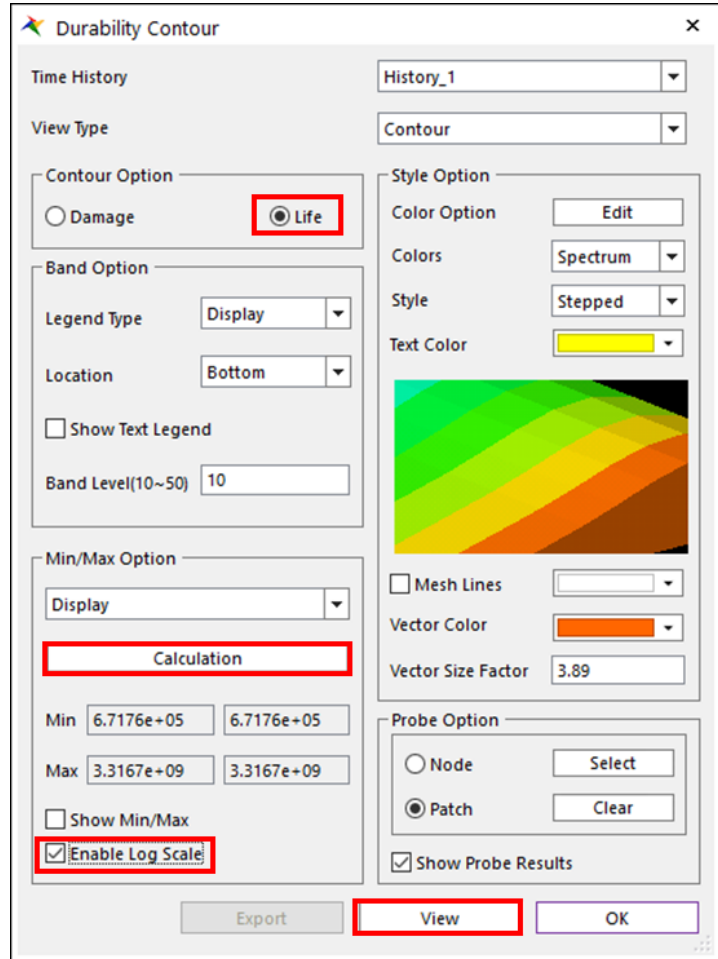
- As shown below, it is possible to check the Stress Time History on the patch which has the maximum damage results among the defined patches.

**To verify the contour results:**

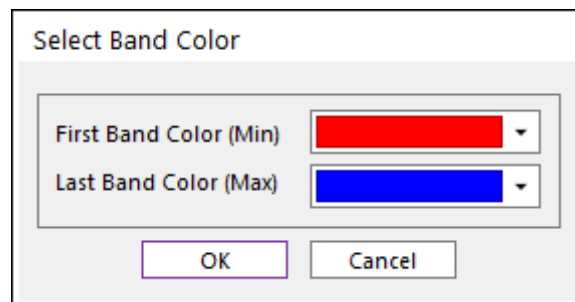
- From the **Durability** group in the **Post Analysis** tab, click **Contour**.

The Durability Contour dialog window appears.

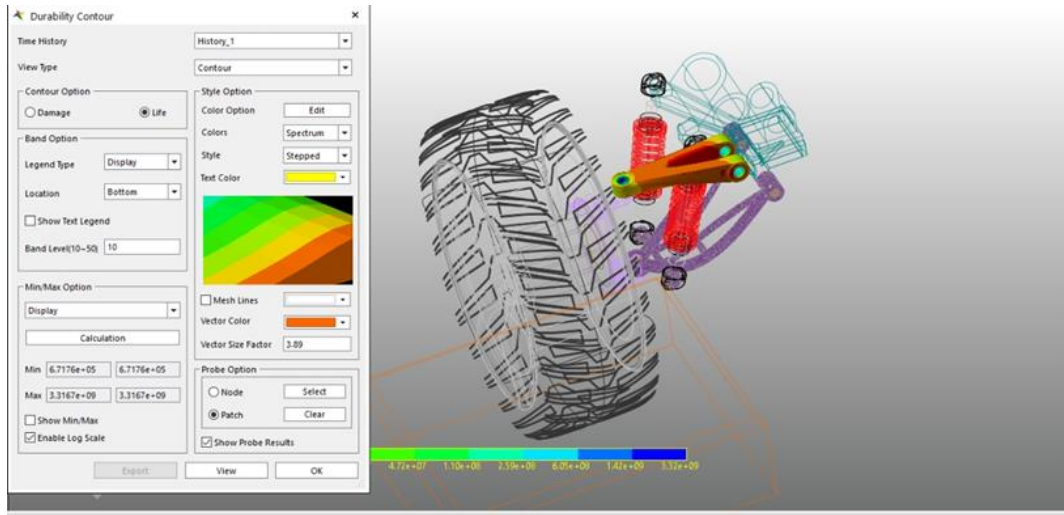
- In the **Durability Contour** dialog window, perform the following:
  - In the **Contour** Option group, select **Life**.
  - Click **Calculation**.
  - Select **Enable Log Scale**.
  - Click **Contour View** to view the results.



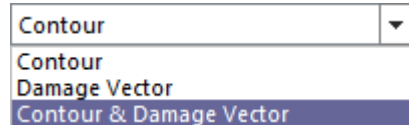
- To make the results more visible, click **Edit** in the **Style Options** group of the Durability Contour dialog window, and change the colors as follows.



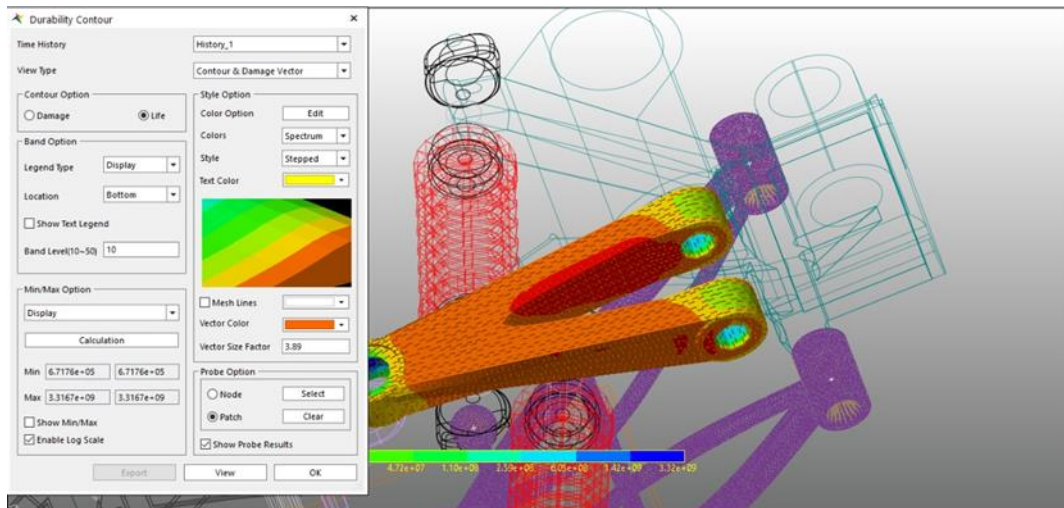
- Click **Contour View** again to highlight the least durable sections in red, as shown below. This makes it easier to identify the areas with a relatively short fatigue life. (**Tip:** At this point, if you would like a more detailed Contour Plot, then select **Wireframe** in the toolbar before viewing the results.)



- Select **Contour & Damage** in the **View Type** on options in order to check the damage direction on each patch.

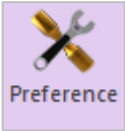


As shown below, if the View button is pushed, the damage direction will appear.

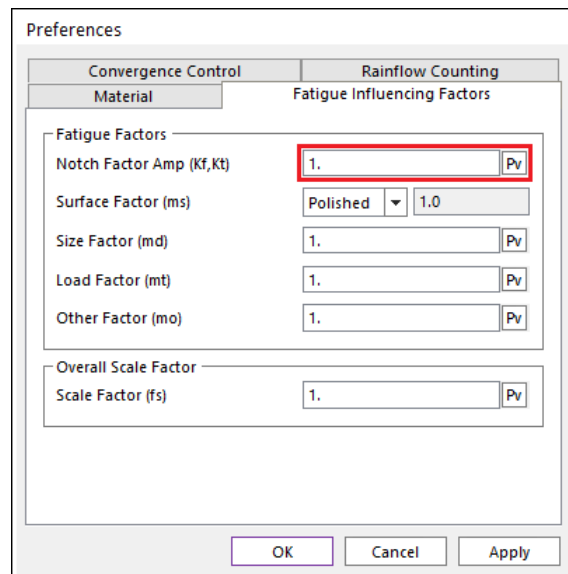


## Conducting a Durability Analysis on an LCA\_FE Body

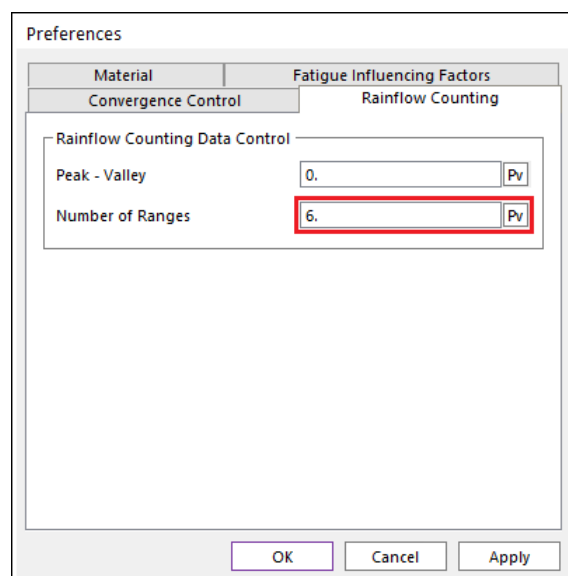
To set the analysis preference:



- From the **Durability** group in the **Post Analysis** tab, click **Preference**.  
The Preference dialog window appears.
- In the Preference dialog window, perform the following:
  - On the Fatigue Influencing Factors tab, in the Fatigue Factors group, change the **Notch Factor Amp (Kf, Kt)** from **2** to **1**.



- On the Rainflow Counting tab, in the Rainflow Counting Data Control, change the **Number of Ranges** to **6**. (**Tip:** At this point, if you change the number of ranges when running Rainflow Counting in the Fatigue Evaluation dialog window and set the number of cycles on mean stress and stress amplitude, then the number of ranges for the mean stress and stress amplitude in the Excel chart will be the value you have set.)



**To use a user-defined S-N curve:**



1. From the **Durability** group in the **Post Analysis** tab, click **Fatigue**.

The Fatigue Evaluation dialog window appears.

2. In the Fatigue Evaluation dialog window, perform the following.
  - Change the **Axial Mode** to **Uni-Axial**.
  - In the **Life Criteria** group, select **User-Defined**.
  - Set the **Searching Increment** item to **10 Deg**.
  - Set the **Occurrence** to **1000000**.

3. In the **Material** group, click "...".

The **User-Defined** dialog window appears.

4. In the **User-Defined** dialog window, perform the following:
  - Click the **Add Row** button **9** times, and enter the values shown in the figure to the right.
  - Click **Draw** to show the **S-N curve** for the given values.
  - Click **Close**.

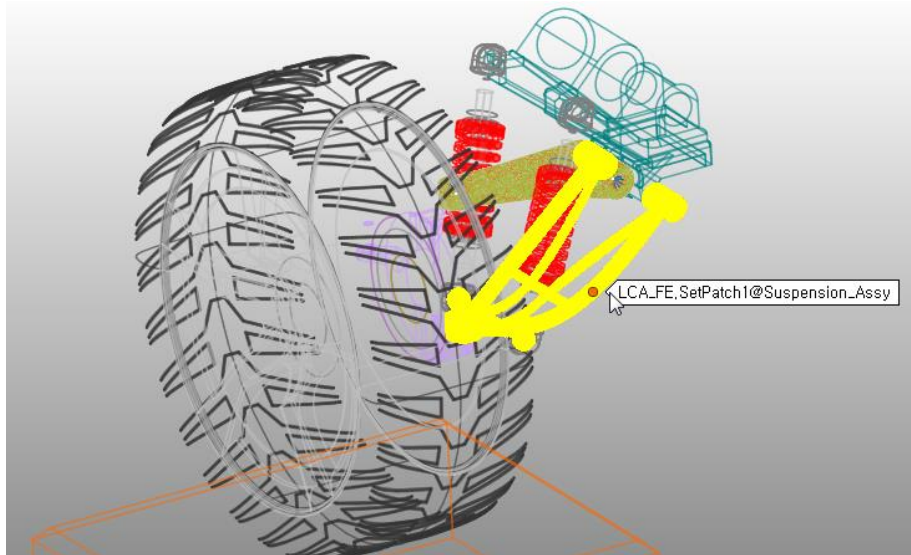
No	Cycle to Failure	Stress Amplitude
1	1	820
2	10	630
3	100	470
4	1000	360
5	10000	270
6	100000	200
7	1000000	150
8	10000000	120
9	100000000	90

5. In the **Element/Patch Set** group, click **EL**.

6. Select **Patch Set** for the **LCA\_FE** body.

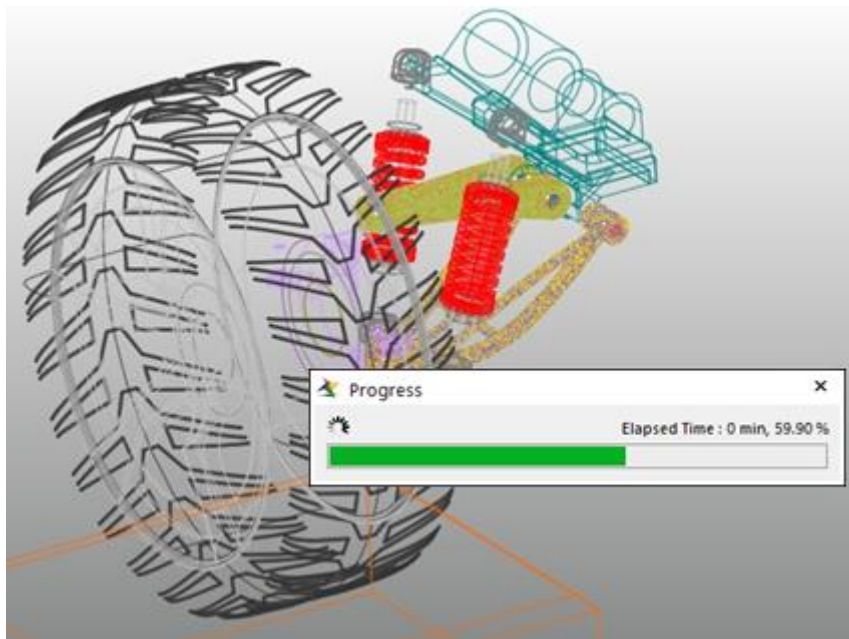


Return to the Fatigue Evaluation dialog window and note that the Element/Patch Set item maintains the patch data of the UCA\_FE body set in the previous step. For this tutorial, however, hold down the **Shift Key** and select the patch data that was defined as the LCA\_FE body, as shown in the figure below.

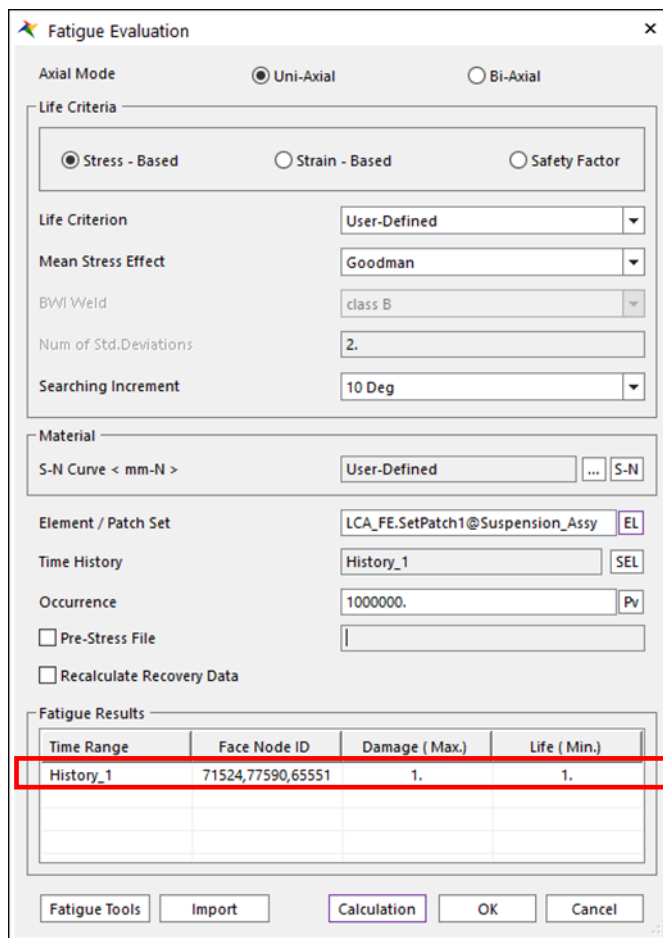


7. Click **Calculation**.

The Fatigue Analysis processing status appears in a Progress Bar, as shown below.

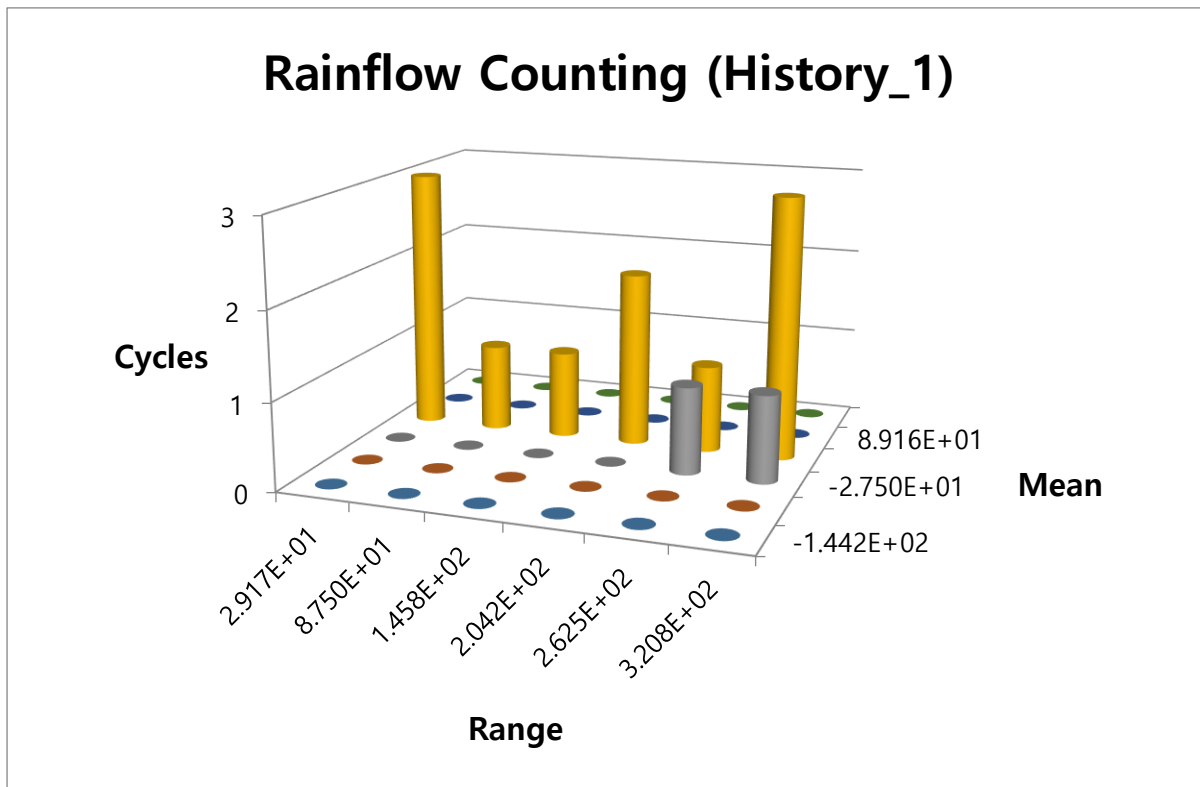


When the Fatigue Analysis is complete, the Fatigue Evaluation dialog window displays the **Maximum Damage** and **Minimum Life**.

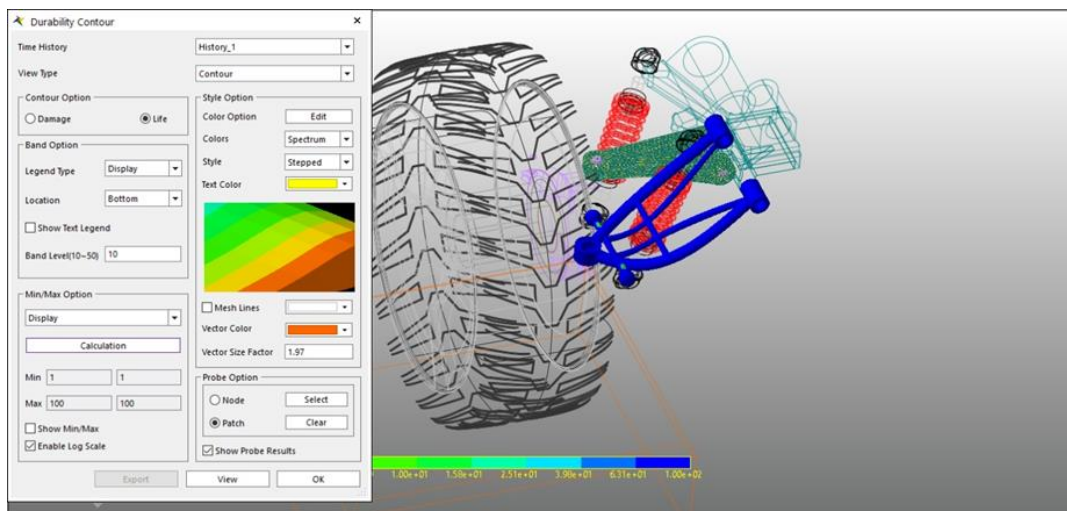


8. In the **Fatigue Evaluation** dialog window, click **Fatigue Tool**.

As shown below, the **Rainflow Counting** results displayed in the Excel file are based on the Stress Time History applied to the patch zone where the damage is the largest. Unlike the **Rainflow Counting** result for the UCA\_FE body, there are 6 ranges in the 3D chart because you changed the number of ranges in the **Rainflow Counting** data set to 6 in the Preference dialog window.



9. Open the **Contour** dialog window, and select **Life** in the **Contour Option** group, as you did when verifying the durability analysis results for the UCA\_FE body.
10. Click **Calculation** and note that the Min/Max values are calculated in the dialog window. Then, click **Contour View** to display the results, as shown below.



Chapter

5

## Analyzing and Reviewing Results

### Task Objective

This chapter teaches you how to analyze and review the durability results of the two models.



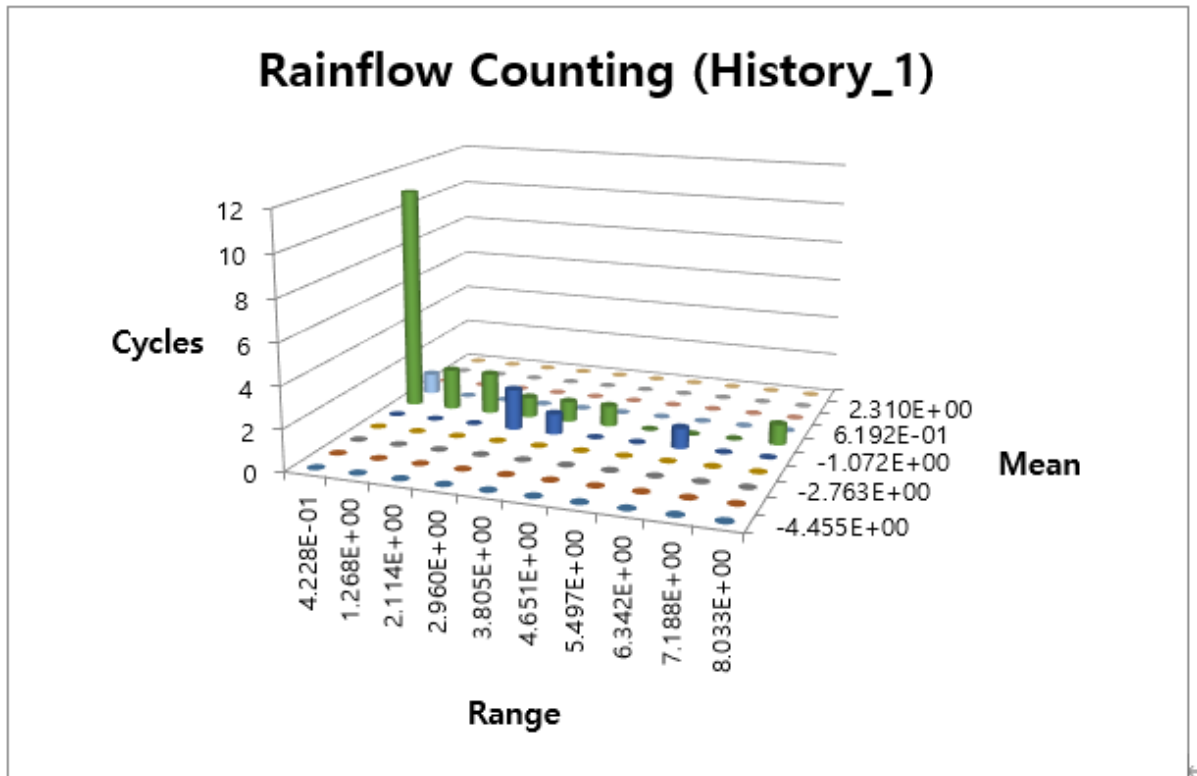
### Estimated Time to Complete

5 minutes

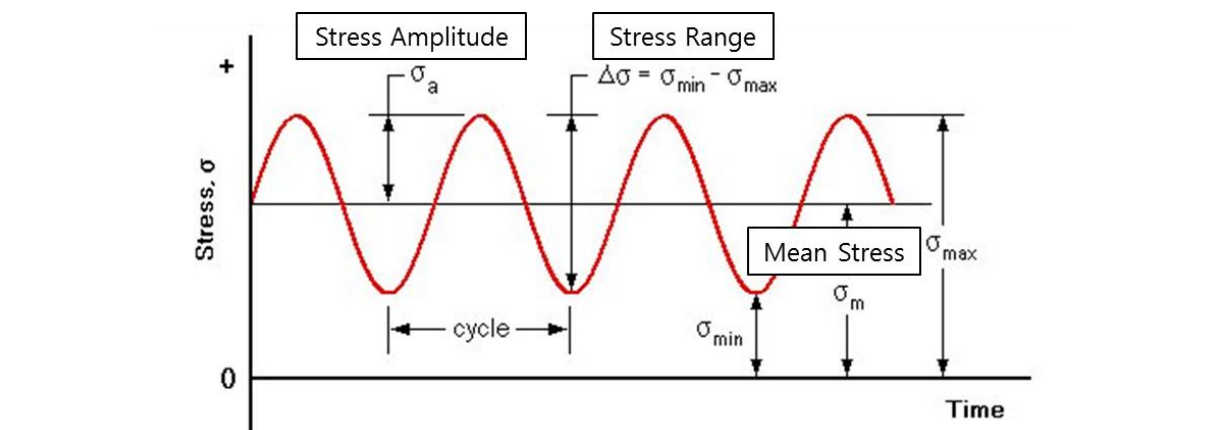
## Analyzing Durability Results

### Analyzing the results for the UCA\_FE body

- Before conducting the durability analysis, if you compared the Von-Mises stresses for the UCA\_FE and LCA\_FE bodies after the final step in Chapter 3, then the stress generated in the UCA\_FE body would be relatively low. Therefore, you would anticipate that its durability would be very high as well. However, In actuality, the rainflow counting result derived from the stress time history used in the fatigue analysis, shown in the figure below, reveals that the largest stress range (where the stress range is twice the stress amplitude) repeatedly added on the weakest point (the patch zone) is about 3 Mpa and the mean stress is less than 1 Mpa.



- For reference, the relationship between the stress amplitude, stress range, mean stress, and cycle are shown in the stress time history below.



- As you can see from the simulation results, the load was applied to the UCA\_FE body repeatedly in the same direction and our durability analysis used the Uni-Axial for the axial type. Since the loads were added a million times during the simulation (occurrence = 1,000,000), this stress life criteria assumes a highly repetitive life. Therefore, the mean stress effect was applied here.
- The durability analysis also used 1020 series steel (available in the Durability Material Library), which is largely used in the vehicle structures.
- The results indicate that the repetition number for the most damaged section, which is the section with the shortest fatigue life, is about  $5e+06$ . In other words, under the load conditions of the simulation, Fatigue Evaluation is the value obtained by multiplying Occurrence condition ( $1e + 06$ ) and fatigue life ( $5e + 06$ ).

**Fatigue Evaluation**

Axial Mode:  Uni-Axial  Bi-Axial

Life Criteria:  Stress - Based  Strain - Based  Safety Factor

Life Criterion: Manson-Coffin

Mean Stress Effect: Goodman

BWI Weld: class B

Num of Std.Deviations: 2.

Searching Increment: 5 Deg

Material: [Steel] 1020 [ Sample.xml ] ... S-N

Element / Patch Set: UCA\_FE.SetPatch1@Suspension\_Assy EL

Time History: History\_1 SEL

Occurrence: 1000000. Pv

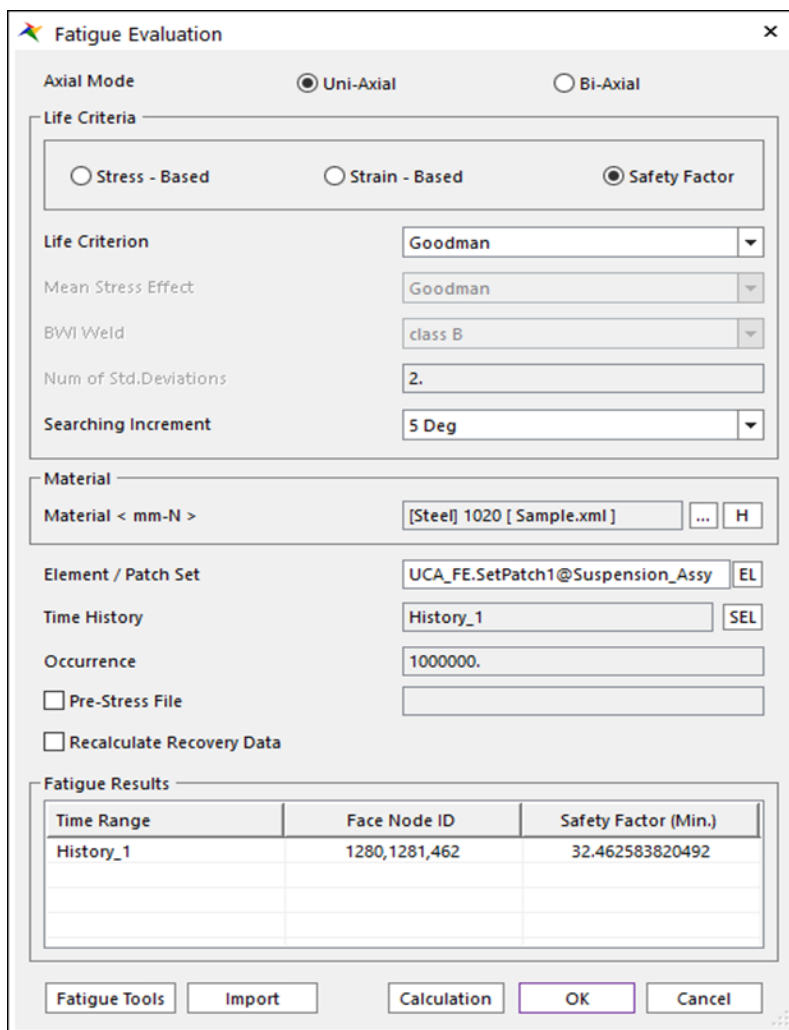
Pre-Stress File

Recalculate Recovery Data

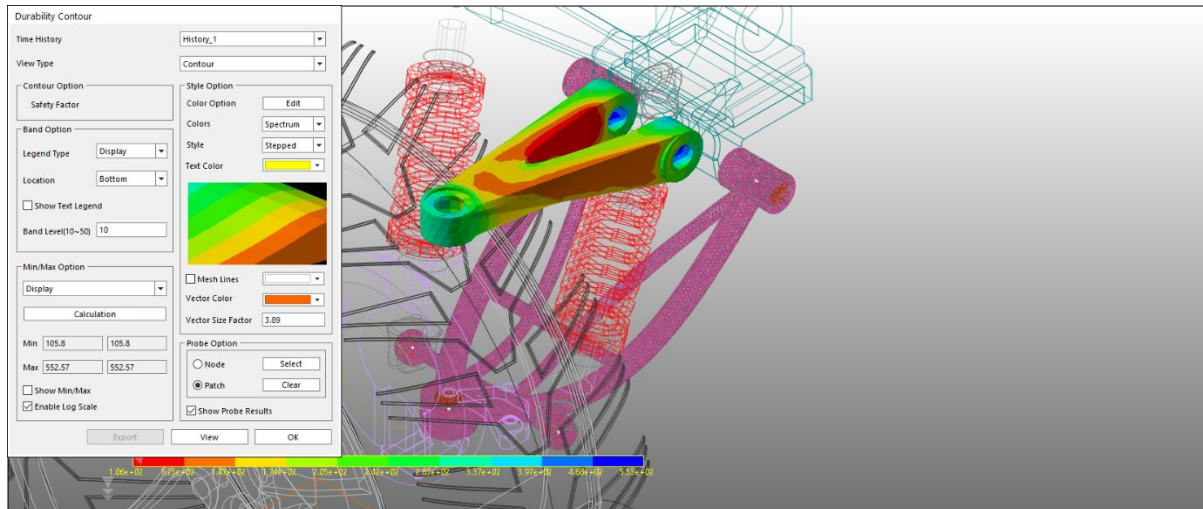
Time Range	Face Node ID	Damage ( Max.)	Life ( Min.)
History_1	1280,1281,462	3.3033604680415...	302722.034024

Fatigue Tools Import Calculation OK Cancel

- Assuming that the fatigue life result shown above has an infinite life, then you can apply the safety factor to the life criteria to derive another fatigue analysis result for the UCA\_FE Body design.
- To do so, select Safety Factor for Life Criteria only, and leave the searching increment, element/patch set, and materials the same as those applied to the UCA\_FE body. This will produce the results shown below. (Tip: At this point, change the notch factor to 2 in the Preference dialog window.)



- Select this option in the **Contour Plot** to produce the following results.

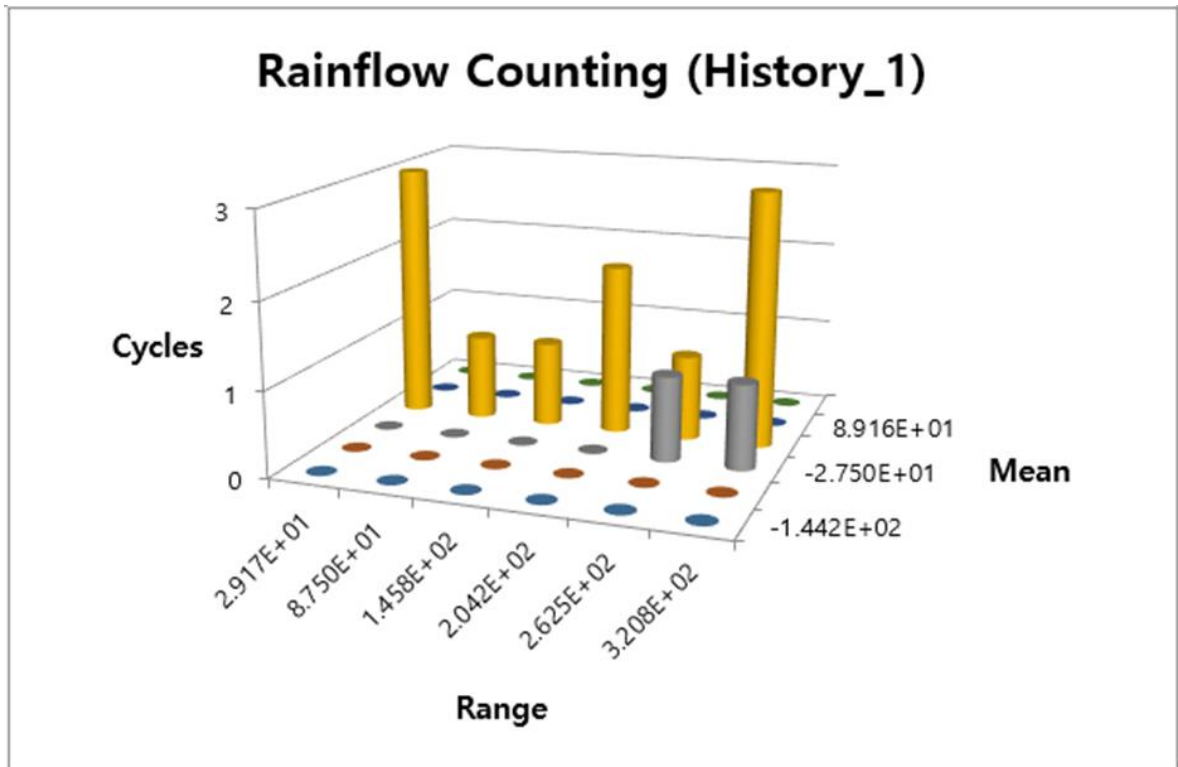


- The conditions required in the Safety Factor calculation are the stress amplitude and the mean stress from the Rainflow Counting results. Therefore, the occurrence data related to the number of cycles is unnecessary. The durability analysis results shown above reveal that the UCA\_FE body is very stable.

### Analyzing the results for the LCA\_FE body

- Unlike the UCA\_FE body, you must enter the formulas for the durability analysis of the LCA\_FE body instead of using the ones provided by RecurDyn/durability, which create specific S-N Curves. This means that you can directly input data obtained from external experiments and/or related resources (stress amplitude vs. cycle) when setting the materials for the fatigue analysis by selecting User-Defined S-N Curve.
- As you can see in the rainflow counting results graph, the stress range and the mean stress are approximately 100 times larger than the results obtained for the UCA\_FE body.
- You can conclude from the fatigue analysis results that the LCA body can survive only up to 1 million times under the given load conditions. Therefore, a design change is essential in order to make the product more durability.
- If you were to determine the stability of the LCA\_FE Body based on the Von-Misess stress only, then the maximum stress would be about 139 Mpa, as shown below, which is far less than the yielded stress (262 Mpa). Therefore, this design could be considered stable. As far as durability is concerned, however, it is necessary to alter the design.





**Fatigue Evaluation**

Axial Mode  Uni-Axial  Bi-Axial

Life Criteria

Stress - Based  Strain - Based  Safety Factor

Life Criterion: User-Defined

Mean Stress Effect: Goodman

BWV Weld: class B

Num of Std.Deviations: 2.

Searching Increment: 10 Deg

Material

S-N Curve < mm-N >: User-Defined ... S-N

Element / Patch Set: LCA\_FE.SetPatch1@Suspension\_Assy [EL]

Time History: History\_1 [SEL]

Occurrence: 1000000. [Pv]

Fatigue Results

Time Range	Face Node ID	Damage ( Max.)	Life ( Min.)
History_1	71524,77590,65551	1.	1.

Fatigue Tools Import Calculation OK Cancel

*Thanks for participating in this tutorial*